

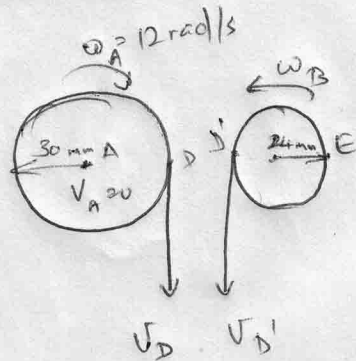
HOMEWORK 10 SOLUTIONS

16.22

$$V_E = 0 \quad V_A = 0$$

$$\vec{\omega}_A = 12\mathbf{k} \text{ rad/s} \quad \vec{\omega}_B = \omega_B \mathbf{k} \text{ rad/s}$$

$$\vec{\omega}_{AB} = \omega_{AB} \mathbf{k} \text{ rad/s}$$



When there is no slip $V_D' = V_D$

$$\begin{aligned} V_D &= V_A + V_{D/A} \\ &= 0 + (\vec{\omega}_A \times r_{D/A}) = -12\mathbf{k} \times 30\mathbf{i} = -360\mathbf{j} \text{ mm/s} \quad \text{--- (1)} \end{aligned}$$

$$V_D' = V_D = V_E + V_{D'/E} = 0 + \vec{\omega}_B \times (-48\mathbf{i}) \quad \text{--- (2)}$$

∴ Equating $-360\mathbf{j} = \omega_B \mathbf{k} \times (-48\mathbf{i})$

$$\Rightarrow \omega_B = \frac{360}{48} = 7.50 \text{ rad/s (ccw)}$$

To calculate ω_{AB} , we need to relate the motion of B to the fixed point A.

V_B can be expressed in terms of POINT E

$$\begin{aligned} \therefore V_B &= V_E + V_{B/E} \\ &= 0 + (\vec{\omega}_B \times r_{B/E}) \quad \text{--- (3)} \end{aligned}$$

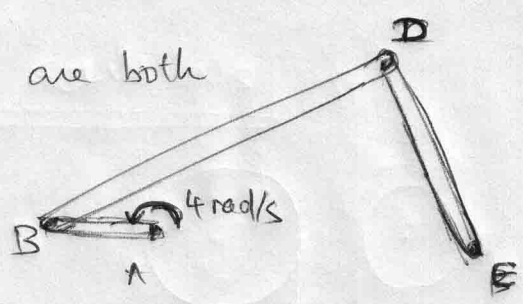
Now V_B in terms of A

$$V_B = V_A + V_{B/A} = 0 + (\vec{\omega}_{AB} \times r_{B/A}) \quad \text{--- (4)}$$

Equating we get $-24\omega_B = 54\omega_{AB} \Rightarrow \omega_{AB} = -3.33 \text{ rad/s (ccw)}$

16.26

Assuming that ω_{DE} & ω_B are both counterclockwise



Relating the point B to A

$$\begin{aligned}
 V_B &= V_A + V_{B/A} \\
 &= 0 + (\vec{\omega}_{BA} \times r_{B/A}) \quad (\Rightarrow V_A = 0) \\
 &= [4k \times (-6i)] = -24j \text{ m/s}
 \end{aligned}$$

Now relating point D to point B, we get

$$\begin{aligned}
 V_B &= V_D + V_{B/D} \\
 \Rightarrow (\vec{\omega}_{AB} \times r_{B/A}) &= V_D + (\vec{\omega}_{BD} \times r_{B/D}) \\
 \Rightarrow -24j &= V_D + (\vec{\omega}_{BD} \times r_{B/D}) \quad \text{--- (1)}
 \end{aligned}$$

Now V_D can be obtained in terms of the point E.

$$\begin{aligned}
 \Rightarrow V_D &= V_E + V_{D/E} \\
 V_E &= 0 \\
 \Rightarrow V_D &= (\vec{\omega}_{DE} \times r_{D/E}) \quad \text{--- (2)}
 \end{aligned}$$

Plugging in (2) in (1)

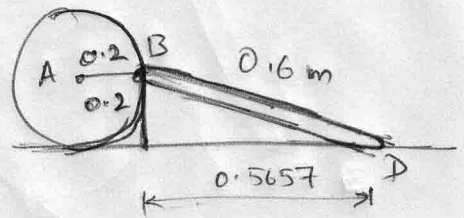
$$\begin{aligned}
 -24j &= (\vec{\omega}_{DE} \times (-6i + 8j)) + [(\vec{\omega}_{BD} \times (-15i - 8j))] \\
 &= (-6\omega_{DE}\hat{j} - 8\omega_{DE}\hat{i}) + (-15\omega_{BD}\hat{j} + 8\omega_{BD}\hat{i})
 \end{aligned}$$

Equating \hat{i} & \hat{j} components & solving gives $\omega_{BD} = \omega_{DE} = 1.143 \text{ rad/s}$

16:30

Given that V_A is to the left

\therefore Assume V_D is directed to the left too.



A is point whose motion is known.

Hence relate B to A.

$$\begin{aligned} V_B &= V_A + V_{B/A} \\ &= -0.6\hat{i} + (\vec{\omega}_{AB} \times r_{B/A}) \end{aligned}$$

Now ω_{AB} can be obtained from dynamics on curvilinear coordinates.

Point B has ω_{AB} such that

$$\omega_{AB} = \frac{V_A}{R} = \frac{0.6}{0.2} = 3 \text{ rad/s}$$

$$\text{Now } V_B = -0.6\hat{i} + (3\hat{k} \times 0.2\hat{i})$$

Now relating D to B

$$V_D = V_B + V_{D/B}$$

$$-V_D\hat{i} = -0.6\hat{i} + (0.6\hat{j}) + (\vec{\omega}_{BD} \times r_{D/B})$$

$$-V_D\hat{i} = -0.6\hat{i} + 0.6\hat{j} + 0.5657\omega_{BD}\hat{j} + 0.2\omega_{BD}\hat{i}$$

Equating \hat{i} & \hat{j} components and solving gives

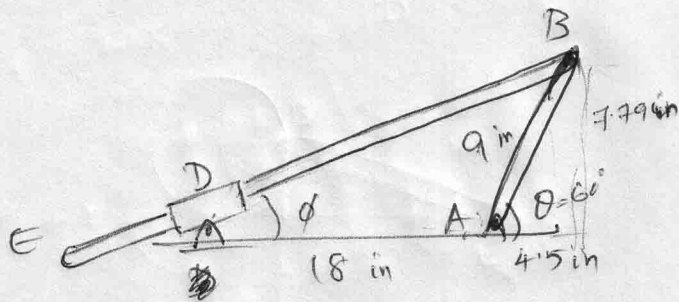
$$\omega_{BD} = -1.0607 \text{ rad/s}$$

$$V_D = 0.812 \text{ m/s.}$$

16-32

Using trigonometry we get

$$\theta = 60^\circ \Rightarrow \phi = 19.11^\circ$$



D is a point on the rod at the collar. Hence D is not a fixed point.

Relating point B to A.

$$\begin{aligned} \vec{V}_B &= \vec{V}_A + \vec{V}_{B/A} \\ &= 0 + (\omega_{AB} \times \vec{r}_{B/A}) \quad (V_A = 0) \\ &= 16 \mathbf{k} \times (4.5 \mathbf{i} + 7.794 \mathbf{j}) \end{aligned}$$

Now relating the point D to E.

[We relate D, as we do not have lengths to associate E and BD has the same motion as BE]

$$\vec{V}_D = \vec{V}_B + \vec{V}_{D/B}$$

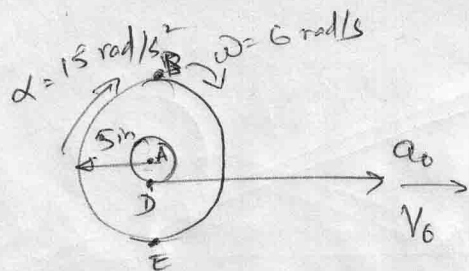
Point D on the rod moves through the collar and hence has a velocity \vec{V}_D directed along the rod.

$$\begin{aligned} \therefore \vec{V}_D [\cos 19.11^\circ \mathbf{i} - \sin 19.11^\circ \mathbf{j}] &= 16 \mathbf{k} \times (4.5 \mathbf{i} + 7.794 \mathbf{j}) \\ &\quad + (\omega_{BD} \times \vec{r}_{D/B}) \\ &= 16 \mathbf{k} \times (4.5 \mathbf{i} + 7.794 \mathbf{j}) + (\omega_{BD} \mathbf{k} \times (-2.5 \mathbf{i} - 7.794 \mathbf{j})) \end{aligned}$$

Equating \mathbf{i} & \mathbf{j} components & solving.

$$\vec{\omega}_{BD} = \vec{\omega}_{BE} = \underline{\underline{4.57 \text{ rad/s}}}$$

16.56



As there is no slip at the point **E**, we can calculate

$$a_A \text{ from } \alpha \text{ as } a_A = R\alpha = 75 \text{ in/s}^2.$$

(a) Relating point D to point A.

$$\begin{aligned} a_D &= a_A + a_{D/A} \\ &= 75 \hat{i} + (\alpha_{AD} \times r_{D/A}) + \omega_{AD} \times (\omega_{AD} \times r_{D/A}) \\ &= 75 \hat{i} + (-15\mathbf{k} \times -2\mathbf{j}) + (-6\mathbf{k}) \times [(-6\mathbf{k}) \times (-2\mathbf{j})] \\ a_D &= 45\mathbf{i} + 72\mathbf{j} \text{ in/s}^2 \end{aligned}$$

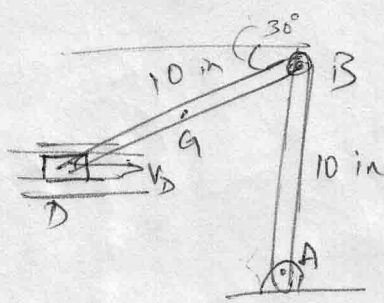
(b) Relating point B to point A

$$\begin{aligned} a_B &= a_A + a_{B/A} \\ &= 75\hat{i} + (\alpha_{AB} \times r_{B/A}) + \omega_{BA} \times (\omega_{AB} \times r_{B/A}) \\ &= 75\hat{i} + (-15\mathbf{k}) \times 5\mathbf{j} + (-6\mathbf{k}) \times [-6\mathbf{k} \times 5\mathbf{j}] \\ a_B &= 75\hat{i} - 180\hat{j} \text{ in/s}^2 \end{aligned}$$

(c) The acceleration at the end of the string equals the horizontal component of acceleration at D = $45 \hat{i} \text{ in/s}^2$.

16.62

Starting from point A we can relate points in order.



Point B

$$V_B = V_A + V_{B/A}$$

At the point shown V_D & V_B are ~~equal~~ parallel as they are both moving horizontally.

$$V_B = 0 + (\omega_{AB} \times r_{B/A}) \quad (V_A = 0)$$

$$\Rightarrow V_B = (\omega_{AB} \times r_{BA}) \quad \text{--- (1)}$$

Relating D to B

$$V_D = V_B + V_{D/B}$$

$$V_D = V_B + (\omega_{DB} \times r_{D/B})$$

As V_D and V_B are parallel, the rod is moving translationally only. $\therefore \omega_{BD} = 0$

$$\Rightarrow \underline{V_D = V_B = 2.4 \text{ ft/s}} \quad \text{--- (2)}$$

Using (2) in (1) we get

$$\omega_{AB} \times (10\hat{j}) = 2.4\hat{i} \Rightarrow \underline{\underline{\omega_{AB} = -2.88 \text{ rad/s}}}$$

Proceeding to acceleration.

$$a_B = a_A + a_{B/A}$$

$$\begin{aligned}
 \Rightarrow a_B &= 0 + \alpha_{AB} \times r_{B/A} + \omega_{AB} \times (\omega_{AB} \times r_{B/A}) \\
 &= \alpha_{AB} \hat{k} \times 10 \hat{j} + (-2.88 \text{ k}) \times [(-2.88 \text{ k}) \times 10 \hat{j}] \\
 &= -10 \alpha_{AB} \hat{i} + (-2.88 \text{ k}) \times (28.8 \hat{i}) \quad \text{--- (3)}
 \end{aligned}$$

Relating D to B

$$\begin{aligned}
 a_D &= a_B + a_{D/B} \\
 &= a_B + (\alpha_{DB} \times r_{D/B}) + \omega_{BD} \times (\omega_{BD} \times r_{D/B}) \\
 &= a_B + [\alpha_{DB} \times (10 \cos 30^\circ \hat{i} + 10 \sin 30^\circ \hat{j})] \quad \text{--- (4)}
 \end{aligned}$$

Using a_B from (3) in (4), and using

$a_D = 0$ as it moves with constant velocity

we get an expression.

Equating i & j components gives

$$\alpha_{BD} = -9.577 \text{ k rad/s}^2$$

For point G $a_G = a_D + a_{G/D}$

$$= a_D + (\alpha_{BD} \times r_{G/D})$$

$$= 0 + (-9.577 \text{ k}) \times (5 \cos 30^\circ \hat{i} + 5 \sin 30^\circ \hat{j})$$

$$= 23.94 \hat{i} - 41.47 \hat{j} \text{ m/s}^2$$

$$a_G = \text{magnitude of } \vec{a}_G = 47.9 \text{ m/s}^2$$

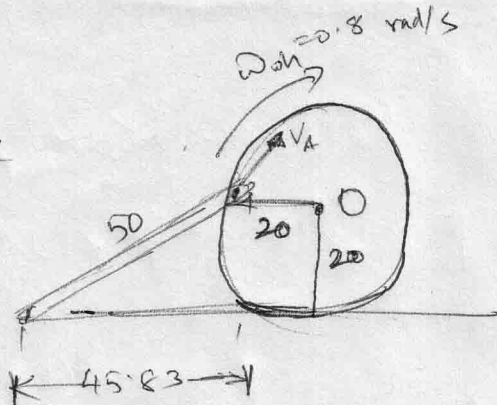
16.66

Using the velocity relations we can relate

$$V_A = V_C + V_{A/C}$$

$$V_C = 0 \text{ (no slipping)}$$

$$\Rightarrow V_A = (\omega_{AC} \times r_{AC}) = 22.63 \text{ in/s}$$



Using instant center method

$$\omega_{AB} = \frac{V_A}{EA} = 0.3492 \text{ rad/s} \quad \text{where E is the instant center}$$

New $a_A = a_O + a_{A/O}$

$$= 0 + \vec{\omega}_{wh} \times (\vec{\omega}_{wh} \times r_{A/O}) \quad \text{--- (1)}$$

$[\alpha_{AO} = 0 \text{ - constant angular velocity}]$

$$a_B = a_A + a_{B/A}$$

$$= \left[\vec{\omega}_{wh} \times (\vec{\omega}_{wh} \times r_{A/O}) \right] + \left[\alpha_{AB} \times r_{B/A} \right] + \left[\vec{\omega}_{AB} \times (\vec{\omega}_{AB} \times r_{B/A}) \right]$$

$$a_B i = (-0.8k) \times (-0.8k \times -20i)$$

$$+ \alpha_{AB} k \times (-45.83i - 20j) + 0.3492k \times (0.3492k \times (45.83i - 20j))$$

$$a_B i = 12.80i - 45.83\alpha_{AB}i + 2\alpha_{AB}i + 5.587i + 2.439j$$

Equating i & j components we can solve for

$$a_B \& \alpha_{AB}$$

Hence $a_B = 19.45i \text{ in/s}^2$

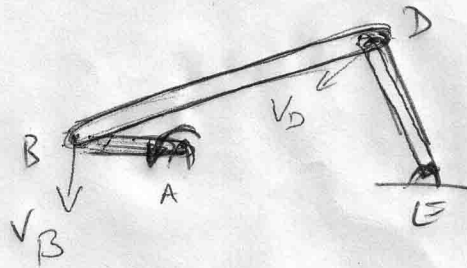
16.68

Using velocity analysis

$$V_B = V_A + (\omega_{AB} \times r_{AB})$$

$$V_A = 0$$

$$\therefore V_B = (4\hat{k} \times -6\hat{i}) = -24\hat{j} \text{ m/s}$$



Using instant center method (which is point E) for both BD & DE

$$\omega_{BD} = \omega_{DE} = 1.143 \text{ rad/s (CCW)}$$

Now relating accelerations

$$a_B = a_A + a_{B/A}$$

$$a_B = 0 + (\alpha_{AB} \times r_{B/A}) + \vec{\omega}_{AB} \times (\vec{\omega}_{AB} \times r_{B/A})$$

$\alpha_{AB} = 0$ — moves with constant angular velocity

$$\therefore a_B = \vec{\omega}_{AB} \times (\vec{\omega}_{AB} \times r_{B/A}) \quad \text{--- (1)}$$

Now relating point D to E

$$a_D = a_E + a_{D/E} \quad (a_E = 0)$$

$$= 0 + (\alpha_{DE} \times r_{D/E}) + \vec{\omega}_{DE} \times (\vec{\omega}_{DE} \times r_{D/E}) \quad \text{--- (2)}$$

Now relating B & D

$$a_B = a_D + a_{B/D}$$

$$a_B = a_D + (\alpha_{BD} \times r_{B/D}) + \vec{\omega}_{BD} \times (\vec{\omega}_{BD} \times r_{B/D}) \quad \text{--- (3)}$$

Plugging in ① & ② into ③ and applying numerical values.

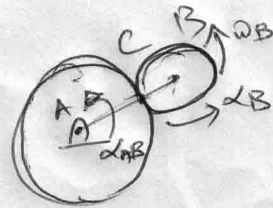
$$0 + 4kx(4kx(-6i)) = \alpha_{DE}k \times (-6i + 8j) + 1143k \times [1.143k \times (-6i + 8j)] + \alpha_{BD}k \times (-15i + 8j) + 1.143k \times [1.143k \times (-15i - 8j)]$$

Equating \hat{i} & \hat{j} components we can solve for the two unknowns α_{BD} & α_{DE}

$$\alpha_{BD} = 2.45k \text{ rad/s}^2 \quad \alpha_{DE} = -6.12k \text{ rad/s}^2$$

16-72

$V_C = 0$ as point of contact doesn't slip.



$$\therefore V_B = V_C + V_{B/C}$$

$$= V_{B/C} = (\omega_B \times r_B) \quad \text{--- ①}$$

Relating A to B

$$V_B = V_A + V_{B/A}$$

$$= 0 + \omega_{AB} \times (r_{B/A})$$

$$= \omega_{AB} \times (r_A + r_B) \quad \text{--- ②}$$

From ① & ②

$$(r_A + r_B) \omega_{AB} = r_B \omega_B$$

$$\Rightarrow \omega_{AB} = \frac{(r_A + r_B) \omega_{AB}}{r_B} = \left(\frac{60 + 25}{25} \right) (5) = 17.0 \text{ rad/s}$$

Time derivative of the above provides

$$\alpha_B = \frac{v_A + v_B}{r_B} \alpha_{AB} = \frac{60 + 25}{25} (12.5) = 42.5 \text{ rad/s}^2 \text{ ccw.}$$

Relating accelerations

$$a_B = a_A + a_{B/A}$$

$$= 0 + (\alpha_{AB} \times r_{B/A}) + \vec{\omega}_{AB} \times (\vec{\omega}_{AB} \times r_{B/A}) \quad (3)$$

Relating C to B

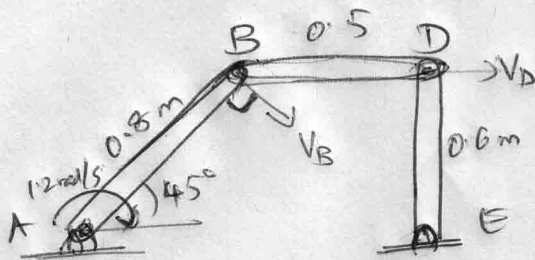
$$a_C = a_B + a_{C/B}$$

$$= a_B + (\alpha_{BC} \times r_{C/B}) + \vec{\omega}_{BC} \times (\vec{\omega}_{BC} \times r_{C/B})$$

Using (3) & plugging in values

$$a_C = (12.5 \mathbf{k} \times 85 \mathbf{j}) + 5 \mathbf{k} \times (5 \mathbf{k} \times 85 \mathbf{i}) + 42.5 \mathbf{k} \times (-25 \mathbf{j}) + 17 \mathbf{k} \times [17 \mathbf{k} \times (-25 \mathbf{j})]$$

$$\therefore a_C = 5.10 \mathbf{i} \text{ m/s}^2$$



16.74 Relating velocities

$$v_B = v_A + v_{B/A}$$

$$= 0 + (\omega_{AB} \times r_{B/A}) = 0.96 \text{ m/s}$$

Using "instant center" method $\omega_{BD} = 1.3577 \text{ rad/s}^2$

$$v_D = 0.67885 \text{ m/s}$$

$$\omega_{DE} = v_D / DE = 1.131 \text{ rad/s (cw)}$$

Relating acceleration

$$\begin{aligned} a_B &= a_A + a_{B/A} \\ &= 0 + (\alpha_{AB} \times r_{B/A}) + \vec{\omega}_{AB} \times (\vec{\omega}_{AB} \times r_{B/A}) \quad \text{--- (1)} \end{aligned}$$

$$\begin{aligned} a_D &= a_E + a_{D/E} \\ &= 0 + (\alpha_{DE} \times r_{D/E}) + \vec{\omega}_{DE} \times (\vec{\omega}_{DE} \times r_{D/E}) \quad \text{--- (2)} \end{aligned}$$

Now relating B & D

$$\begin{aligned} a_D &= a_B + a_{D/B} \\ &= a_B + (\alpha_{DB} \times r_{D/B}) + \vec{\omega}_{BD} \times (\vec{\omega}_{BD} \times r_{D/B}) \quad \text{--- (3)} \end{aligned}$$

Using (1) and (2) in (3) and plugging in values

$$\begin{aligned} \alpha_{DE} k \times 0.6j + (-1.131k) \times [(-1.131k) \times 0.6j] \\ = 0 + (-1.2k) \times [(-1.2k) \times (0.5657i + 0.5657j)] \\ + \alpha_{BD} k \times 0.5i + 1.358k \times (1.358k \times 0.5j) \end{aligned}$$

Equating i components we can solve for

α_{DE}

$$\alpha_{DE} = 2.894 \text{ rad/s}^2 \quad (\text{ccw})$$

$$\begin{aligned} \therefore a_D &= -0.6(2.894i - 0.7675j) \\ &= -1.736i - 0.7675j \text{ m/s}^2 \end{aligned}$$